

Reconciling Indoor Air Quality and Energy Efficiency in Air-Conditioned Classrooms: Case Studies for Portugal and Macao

Rogério Duarte and Ingride Beltrão-Coelho

Abstract—Growing awareness on the subject of classroom's indoor air quality and concerns regarding the probable link between indoor air quality, student's attendance and academic performance, support recent legislation limiting classroom's indoor pollutant concentrations. A consequence of this legislation is the generalized shift towards air-conditioned classrooms and increased budgets for schools energy running costs. According to the scientific literature, demand control ventilation systems are suited for classrooms, ensuring indoor air quality and thermal comfort with significant energy savings. Based on detailed building energy simulation this paper compares a demand control ventilation system and a (traditional) constant air volume system considering a typical classroom located in two distinct climatic regions: Portugal and Macao. Based on hourly results of classroom's indoor CO₂ concentrations and indoor air temperatures, and on annual air-conditioning energy consumptions, advantages of the demand control ventilation system are discussed.

Index Terms—Classroom indoor air quality, demand control ventilation, energy efficiency, thermal comfort.

I. INTRODUCTION

Several European governments have decided to invest in rebuilding and refurbishing their stock of existing public schools. According to Mumovic [1], UK's government is committed to a £45 billion program dedicated to the rebuilding of secondary schools in England and Wales. Gertis [2] states that during the current decade, Germany will invest €73 billion in the rebuilding and refurbishing of public school buildings. The Portuguese government has also invested in the rebuilding of over 100 existing secondary schools [3] and a trend that favors cyclic rebuilding instead of new construction is foreseeable in Europe, in the future.

These large scale public investments take place at a time of growing awareness on the subject of classroom's indoor air quality (IAQ) [4], [5] and of concern regarding the probable link between IAQ and student attendance and academic performance [6]-[10].

International standards [11], [12] state limits to the maximum concentrations of indoor air pollutants, and European Member States legislate on the subject [13]. Southwest Asian countries share similar concerns on IAQ,

having different indoor air quality and green building labeling methods available.

Traditionally, European classrooms were naturally ventilated; in warmer Mediterranean countries classrooms were often neither heated nor cooled. However, the need to ensure indoor pollutant concentrations below specific maxima imposes the use mechanical ventilation; therefore all rebuilt and refurbished schools now include heating and cooling systems, used to remove the fresh air thermal loads and remove the classroom thermal loads.

These "new" retrofitted schools, with air-conditioned classrooms, represent a "paradigm shift" in Europe, especially for Mediterranean countries. An important consequence of this "shift" is being felt at school administration boards, which are now facing a significant increase in schools energy running costs. The use of energy efficient air-conditioned systems is essential to minimize the problems associated with excessive increase in energy running costs.

Macao and southwest Asian conditions are very different from those in Europe; nevertheless, European design guidelines are a strong influence in Macao's and Hong-Kong's building construction¹. Due to the hot and humid climate, classrooms of southwest Asia are typically air-conditioned, and the associated energy running costs are accepted as the price to pay for indoor thermal comfort. According to [14], a significant amount of Hong Kong's building sector electricity costs is related to air-conditioning. To reduce the air-conditioning payload there is the need to study the use of energy efficient air-conditioned systems: systems that ensure indoor air quality, thermal comfort, and, simultaneously, reduce energy consumption.

This paper first defines a typical classroom that meets contemporary international standards. The use of a Demand Control Ventilation (DCV) system that modulates the fresh airflow rate into the classroom based on indoor CO₂ concentration measurements is then introduced. The modeling of the classroom and HVAC system is described for two distinct locations: Portugal, representing the Mediterranean mild climate, and Macao, representing the hot and humid climate of southwest Asia. Finally, results are presented and the advantages of the DCV system are discussed.

II. TYPICAL CLASSROOM

The classroom that will be used in this study has the

1. For Macao, especially after the nineteen-eighties.

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geometry presented in Fig. 1 and represents a typical 25-30 students classroom. With approximately 50 m² floor surface, has comfortable headroom, generous window area facing south² and a depth (6 m) that does not exceed three times the height of the windows. Access to the classroom is made through a door to a north facing daylighted corridor that is also used to access contiguous classrooms (to the east and west). The typical classroom is located in an intermediate floor.

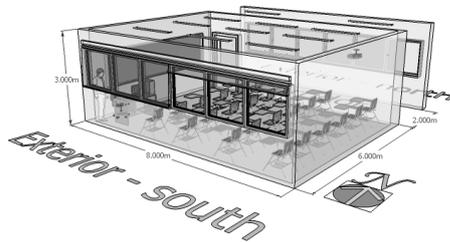


Fig. 1. Typical classroom.

Table I presents further classroom characteristics. When justified, distinctions are made between Macao (MAC) and Portugal (PT). It is important to emphasize that some of the characteristics may not represent the most common ones at each location; the objective is to study a retrofitted classroom with updated equipments and building materials.

TABLE I: RETROFITTED CLASSROOM'S CHARACTERISTICS

Characteristic	Description
Window glazing	Double 3mm-13mm-3mm; clear (PT) Single 6mm; tinted (MAC)
Window frame	Metal framing
Solar control	Exterior light colored shades (PT) Interior light colored shades (MAC)
Lighting	Fluorescent lamps
Equipments	Interactive blackboard

III. DEMAND CONTROL VENTILATION

Whenever a room has a high occupancy density – such as classrooms – typical design fresh airflow rates are usually enough to remove the room thermal loads. For mild climates, such as that of Mediterranean countries, removing the outdoor air thermal loads³ can be enough to provide thermal comfort. For the hot and humid climate of Macao removing the outdoor air thermal loads is not enough and the supply air is typically 8-15 K below the indoor temperature set point.

But actual classrooms occupancy patterns can be quite different from the ones typically used when designing Heating, Ventilating and Air-Conditioning (HVAC) systems. Classrooms occupancy may change due to absenteeism, breaks between classes, field trips, etc. The opening of doors (or windows) also has a significant influence on the classroom ventilation and, IAQ. A traditional Constant Air Volume (CAV) system will continue to supply a fixed amount of fresh air to the classrooms, regardless of how they are being used. A DCV system has the advantage of varying

the airflow rate to every classroom (or group of classrooms) depending on the indoor CO₂ concentration in the classrooms. If a classroom is at its maximum occupancy, CO₂ will rise rapidly and the DCV system will modulate the fresh airflow rate to a maximum. However, if the occupancy decreases, the CO₂ concentration will also decrease, and the DCV system will reduce the fresh airflow rate so that IAQ is ensured and the indoor air temperature set point is kept.

Scientific and technical literature discusses the benefits of DCV systems [15] - [18]. According to [17] implementing DCV systems in schools can save from 20% to 40% of the air-conditioning energy consumption. This argument has certainly contributed to the frequent use of DCV systems in USA school buildings.

Fig. 2 presents a schematic diagram of a DCV system. Variable Air Volume (VAV) boxes can be modulated to control the airflow rate supplied to each classroom. CO₂ concentration inside the classroom is measured and a control system commands actuators that open or close the VAV box dampers. An Air Handling Unit (AHU) is used to filter, heat or cool (and dehumidify) the outdoor air. The AHU also includes an exhaust air heat recovery module.

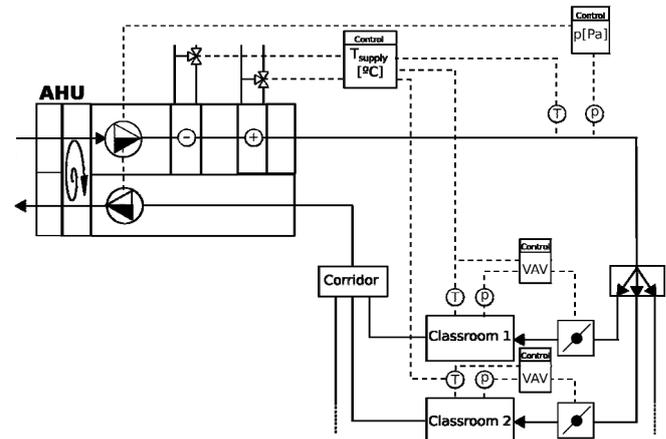


Fig. 2. Schematic diagram of a DCV airflow network, representing VAV boxes, AHU, sensors (pressure, temperature, CO₂), and actuators.

IV. MODELING

EnergyPlus version 6-0-0 [19] was used to model the classroom geometry and the characteristics presented in section II. Hourly results describing the classroom IAQ and thermal behavior were obtained considering DCV and CAV systems, luminance and solar control systems.

The TRY weather files for Lisbon (Portugal) and Macao [20] were used to represent hourly averaged outdoor conditions.

A. Internal Gains and Schedules

Table II presents specific power released in the classroom due to occupancy, artificial lighting and equipments.

Classroom was assumed occupied in working days from 15 September to 15 June, from 8 AM to 12 PM and 13 PM to 18 PM, with two fifteen days holiday periods during Christmas and Easter. Classes last for 90 minutes with 15 minutes breaks. This same schedule was used for equipment. Regarding artificial lighting, for cleaning purposes lights were turned on one hour earlier, at 7 AM.

² For Macao's hot climate, instead of reducing the winter heating loads, the south facing windows increase the classroom's cooling loads. This orientation is justified only if adequate solar and daylighting control devices are used.

³ Cooling or heating the outdoor air to a supply temperature similar to the indoor setpoint temperature, between 20°C and 26°C.

TABLE II: CLASSROOM'S INTERNAL GAINS

Classroom	Internal gain
Occupancy (total gains)	0.45 person/m ² (PT); 0.50 person/m ² (MAC); 86.25 W/person
Artificial lighting	8.3 W/m ²
Equipment (sensible gains only)	12.5 W/m ²

B. Systems' Setpoints

Category II of standard EN15251:2007 [11] was used in defining the setpoints for luminance and solar control systems and for the HVAC systems – see Table III.

TABLE III: SYSTEMS' SETPOINTS (BASED ON CATEGORY II OF EN15251:2007)

System	Control variable	Setpoint
Daylighting control	Luminance [lux] (at the working plane)	300
Solar control	UGR glare index	19
HVAC (DCV)	CO ₂ concentration [ppm] above outdoor air concentration	500 ^(a)
HVAC (DCV & CAV)	Min. operative temperature [°C] Winter season (1 clo)	20
HVAC (DCV & CAV)	Max. operative temperature [°C] Summer season (0.5 clo)	26

a) Assuming an outdoor CO₂ concentration of 400 ppm, the maximum classroom CO₂ concentration becomes 900 ppm.

Luminance setpoint was defined at desk surface. Two stage (150 lux each) artificial lighting was considered. When daylighting alone produces luminance levels higher than 300 lux, artificial lighting is switched off. When daylighting is not enough to obtain 150 lux at desk surface, both stages of artificial lighting are switched on.

Solar control is made possible by modulating the position of the shading devices. The control variable used is the UGR glare index [11] measured at the blackboard plane. Solar control system authority is higher than that of the daylighting control system.

The CO₂ concentration setpoint is available only for the DCV system and the IAQ control is ensured by the VAV boxes in Fig. 2. Both the DCV and the CAV systems control the indoor air temperature⁴. The CAV system varies the supply air temperature with a constant fresh airflow rate (8 l/s/occupant). The DCV system also varies the supply air temperature, and, if necessary, increases the fresh airflow rate (above what is necessary to ensure IAQ) until a maximum equal to the CAV system constant airflow rate.

When modeling the Portuguese climate, supply air temperatures varied between 20 °C and 23°C (implementing a system that is limited to removing the outdoor air thermal loads). For the hot and humid climate of Macao, lower supply air temperatures were allowed (minimum of 14°C)⁵.

The HVAC's system modeling was validated by checking that:

- 1) Supply airflows to classroom were the ones set for the CAV system; and did not exceed the maximum or minimum set for the DCV system;

⁴ For the DCV system, IAQ control authority is higher than that of indoor air temperature control.

⁵ For Macao, thermal comfort requires the removal of classroom thermal loads. Since these loads vary among classrooms (because of differences in orientation, for example), classrooms with similar loads are grouped and connected to specific AHU.

- 2) Heat recovery of exhausted air was correctly computed, given the outdoor and exhaust air conditions;
- 3) The behavior of the DCV system correctly took into account the indoor CO₂ concentration setpoint;
- 4) The indoor air temperatures were consistent with the implemented control laws.

Fig. 3 presents a detail of the output of EnergyPlus for a hot summer day, when the typical classroom is modeled with the DCV system. Lines for hourly supply airflow rate (m³/s) and CO₂ concentration levels (ppm) in the classroom are shown.

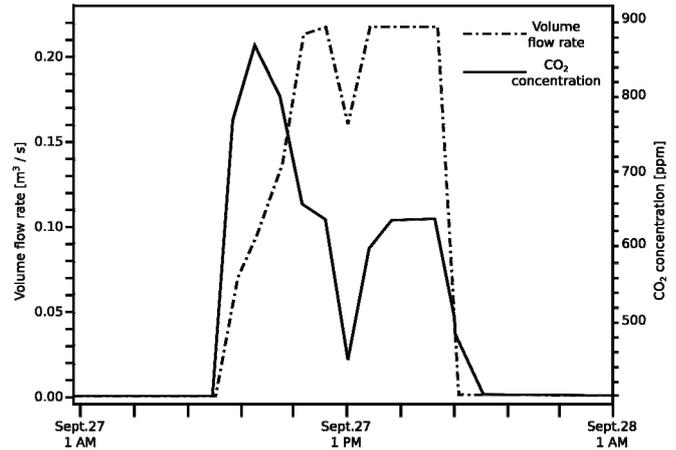


Fig. 3. Hourly airflow rate and CO₂ concentration in a hot summer day: Example of DCV system control.

Fig. 3 shows a significant increase in the classroom's CO₂ concentration at the beginning of the occupancy period, with a resulting increase in the airflow rate supplied to the classroom, necessary to prevent indoor CO₂ concentrations in excess of 900 ppm. Because of the increasing outdoor air temperature (which exceeds 30°C by the end of the morning), airflow rate continues to increase, not because of IAQ, but to maintain indoor air temperatures lower than the maximum setpoint (26°C). This justifies the low 640 ppm CO₂ concentration, below the 900 ppm setpoint. As expected, during the lunch break the DCV system reduces the fresh airflow rate.

Regarding infiltration / natural-ventilation into the classroom, the "DOE2 model" described at Energy Plus manual [19] was considered. Whenever mechanical ventilation is switched on infiltration flow rates were reduced by 75%.

V. RESULTS

EnergyPlus results were used to compare the performance of DCV and CAV systems. For Portugal both systems removed only the outdoor air thermal loads and the range of supply air temperatures (20 - 23°C) supports freecooling. For Macao, supply air temperatures range from the 20°C required to remove outdoor air thermal loads in colder mornings, to the 14°C necessary to ensure comfort indoor air temperatures during the rest of the day.

For the occupancy schedule presented in section IV.A, a total of 1740 hours were modeled per academic year. The hourly results were combined and presented as:

- 1) Means, standard deviations, maxima and minima;

- 2) Percentage of hours with indoor temperature (T_{in}) below a minimum ($T_{min}=19.5^{\circ}\text{C}$) and average deviation from this limiting value – see (1) and (2);
- 3) Percentage of hours with indoor temperature above a maximum ($T_{max}=26.5^{\circ}\text{C}$) and average deviation from this limiting value;
- 4) Percentage of hours with indoor CO_2 concentration above a maximum ($c_{max}=950$ ppm) and average deviation from this limiting value.

Equations (1) and (2)⁶ are presented for item (ii) above; equivalent expressions were used for items (iii) and (iv).

$$\text{Hours with } T_{in} < T_{min} [\%] = \frac{\sum_{i=1}^n H(T_{min} - T_{in,i})}{n} \quad (1)$$

$$\begin{aligned} \text{Mean deviation} \\ \text{from } T_{in} = T_{min}, \\ \text{when } T_{in} < T_{min} [^{\circ}\text{C}] = \frac{\sum_{i=1}^n (T_{in,i} - T_{min})H(T_{min} - T_{in,i})}{\sum_{i=1}^n H(T_{min} - T_{in,i})} \end{aligned} \quad (2)$$

TABLE IV: RESULTS USING DCV AND CAV SYSTEMS IN PORTUGAL.

		Outdoor Environment		Indoor		
		DCV system	CAV system	DCV system	CAV system	
Infiltration [a.c.h.]	Average	-	0.4	0.4		
	Standard deviation	-	0.1	0.1		
	Maximum	-	0.8	0.8		
	Minimum	-	0.0	0.0		
Mechanical ventilation [a.c.h.]	Mean	-	4.0	5.6		
	Standard deviation	-	1.4	0.1		
	Maximum	-	5.8	5.8		
	Minimum	-	1.6	5.4		
Temperature	Mean [$^{\circ}\text{C}$]	16	25	25		
	Standard deviation [$^{\circ}\text{C}$]	5	2	2		
	Maximum [$^{\circ}\text{C}$]	32	31	31		
	Minimum [$^{\circ}\text{C}$]	4	18	18		
	Hours with $T_{in} < T_{min}$ [%]	80	0	0		
	Hours with $T_{in} > T_{max}$ [%]	5	16	15		
	Mean deviation when $T_{in} < T_{min}$ [$^{\circ}\text{C}$]	-6	-	-		
	Mean deviation when $T_{in} > T_{max}$ [$^{\circ}\text{C}$]	2	1	1		
	CO₂ concentration	Mean [ppm]	400	770	647	
		Standard deviation [ppm]	0	115	18	
Maximum [ppm]		-	975	677		
Minimum [ppm]		-	604	603		
Hours with $c_{\text{CO}_2} > c_{\text{max}}$ [%]		0	1	0		
Mean deviation when $c_{\text{CO}_2} > c_{\text{max}}$ [ppm]		-	17	-		
Specific energy consumption	Heating [$\text{W}_{\text{thermal}}/\text{m}^2$]	-	1.3	6.6		
	Cooling [$\text{W}_{\text{thermal}}/\text{m}^2$]	-	1.9	1.9		
	Ventilation [$\text{W}_{\text{electricity}}/\text{m}^2$]	-	1.9	2.7		
	Total [$\text{W}_{\text{electricity}}/\text{m}^2$] (a)	-	2.8	5.0		

(b) Considering EER = 3 and COP = 4 for the production of cold and hot water.

Tables IV and V present classroom's CO_2 concentrations, indoor air temperatures, and air-conditioning systems energy consumptions, for Portugal and Macao, respectively.

The results show that:

- 1) Regarding IAQ, DCV and CAV systems perform equally well.
- 2) For Portugal, with the DCV system, CO_2 concentrations exceed the 950 ppm limit during 1% (17 hours) of the annual occupancy period. However, the average

⁶ H represents the left-continuous Heaviside function.

- deviation from the 950 ppm limit is 17 ppm.
- 3) Due to the higher occupancy (see Table II), for Macao, and with the DCV system, CO_2 concentrations exceed the 950 ppm limit during 16% (278 hours) of the annual occupancy period. However, the average deviation from the 950 ppm limit is just 33 ppm and the maximum CO_2 concentration hardly exceeds 1000 ppm.
- 4) Regarding indoor air temperatures, DCV and CAV systems perform equally well.
- 5) Regarding energy consumption, the DCV enables significant energy savings.
- 6) For Portugal, the DCV system allows (total) electricity savings of approximately 40%; with 80% less electricity spent for heating (COP=4) and 30% less electricity spent for ventilating.
- 7) For Macao, the DCV system (total) electricity saving amounts to just 10%. This is related to the uniform load/occupancy patterns imposed on the DCV system, which runs for longer time periods closer to nominal conditions.

TABLE V: RESULTS USING DCV AND CAV SYSTEMS IN MACAO.

		Outdoor Environment		Indoor		
		DCV system	CAV system	DCV system	CAV system	
Infiltration [a.c.h.]	Average	-	0.3	0.3		
	Standard deviation	-	0.1	0.1		
	Maximum	-	0.8	0.8		
	Minimum	-	0.0	0.0		
Mechanical ventilation [a.c.h.]	Mean	-	4.5	5.6		
	Standard deviation	-	1.2	0.1		
	Maximum	-	5.7	5.7		
	Minimum	-	1.8	5.4		
Temperature	Mean [$^{\circ}\text{C}$]	21	25	25		
	Standard deviation [$^{\circ}\text{C}$]	5	1	2		
	Maximum [$^{\circ}\text{C}$]	33	29	29		
	Minimum [$^{\circ}\text{C}$]	10	21	20		
	Hours with $T_{in} < T_{min}$ [%]	42	0	0		
	Hours with $T_{in} > T_{max}$ [%]	23	16	16		
	Mean deviation when $T_{in} < T_{min}$ [$^{\circ}\text{C}$]	-4	-	-		
	Mean deviation when $T_{in} > T_{max}$ [$^{\circ}\text{C}$]	2	1	1		
	CO₂ concentration	Mean [ppm]	400	770	680	
		Standard deviation [ppm]	0	114	20	
Maximum [ppm]		-	1030	705		
Minimum [ppm]		-	626	622		
Hours with $c_{\text{CO}_2} > c_{\text{max}}$ [%]		0	16	0		
Mean deviation when $c_{\text{CO}_2} > c_{\text{max}}$ [ppm]		-	33	-		
Specific energy consumption	Heating [$\text{W}_{\text{thermal}}/\text{m}^2$]	-	0.1	0.9		
	Cooling [$\text{W}_{\text{thermal}}/\text{m}^2$]	-	81.5	88.4		
	Ventilation [$\text{W}_{\text{electricity}}/\text{m}^2$]	-	2.0	2.7		
	Total [$\text{W}_{\text{electricity}}/\text{m}^2$] (a)	-	29.2	32.3		

(c) Considering EER = 3 and COP = 4 for the production of cold and hot water.

VI. CONCLUSION

The results show that the DCV system ensures IAQ and comfort indoor air temperatures with fresh airflow rates lower than typical design standards (8 l/s/person). The energy saving potential of the DCV system is related to the probability of partial load/occupancy. For the Portuguese mild Mediterranean climate – for which classroom thermal comfort is possible by simply removing the fresh air thermal loads –, winter conditions promote extended periods of time

at partial load operation, with significant energy savings due to reduced ventilation rates (without compromising IAQ). During summer, solar and internal gains push air-conditioning systems to full load operating conditions, justifying Tables IV and V lower saving opportunities associated with cooling.

It should be emphasized that the results presented are for a classroom with generous window areas facing south, and with a uniform occupancy pattern. The electricity saving potential of DCV systems should increase above 10% for cooling, if the classroom orientation and occupancy pattern promote wider daily and annual thermal-load/occupancy oscillations.

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